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ANALYSIS OF FREE OSCILLATIONS OF ROUND THIN PLATES OF VARIABLE THICKNESS WITH A POINT SUPPORT (p. 6–12)**Kirill Trapezon**National Technical University of Ukraine
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This paper reports the derived general analytical solution to the IV-order differential equation with variable coefficients for the problem on free axisymmetric oscillations of a circular plate of variable thickness. The plate thickness changes along the radius ρ in line with the parabolic law $h=H_0(1-\mu\rho)^2$. When building a solution, the synthesis of the factorization method and the symmetry method was used. The factorization method has enabled us to represent the solution to the original IV-order equation as the sum of the solutions to the two respectively constructed II-order equations. The method of symmetry has produced precise solutions to these two equations.

The problem on a point fixation of the plate has been considered as a boundary case of the problem on the rigid fixation of the inner contour of a circular plate whose $\rho \rightarrow 0$. To this end, the general solution has been transformed into the form that pre-meets the conditions on a rigid point support. The result of such a transformation is a simpler solution, with only two permanent integration variables instead of four. As a result, the frequency equation for a plate under any conditions on the outer contour is significantly simplified because it is derived from the second-order determinant. The frequency equation for a plate with a point support and with a free edge at $\mu=1.39127$, which corresponds to the ratio of the limit thicknesses equal to 10.8, yielded the first five eigenvalues λ_i ($i=1\div 5$). The oscillation shapes have been constructed as a graphic illustration for λ_i ($i=1\div 3$). The numerical values of amplitude ratios have been given, as well as coordinates (relative radii) of the oscillation antinodes and nodal circles for each of the five oscillation shapes ($i=1\div 5$). The derived numerical values of the oscillation parameters could in practice be used to initially identify the type of an oscillatory system and its possible characteristics for the case when a plate is fixed inside the inner contour of the small diameter. The criterial ratio of the fastening contour diameter to the plate diameter could serve the same purpose. If this ratio is equal to or less than 0.2, then it is permissible to assume that it is a point-type fastening. In this case, it is possible to calculate the circular plate oscillations with its internal contour fixed using the algorithm set out for a plate with a point support.

Keywords: free oscillations, symmetry method, thin plate, point support, analytical solution.

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THE HOMOGENIZATION OF MULTI-MODULAR COMPOSITES AT THEIR LONGITUDINAL DEFORMATION (p. 13–19)

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A model has been proposed of the homogenization of a transversally-isotropic composite material, whose mechanical characteristics differ at the longitudinal stretching and compression. On its basis, the longitudinal elasticity module of the first kind has been derived, as well as a Poisson coefficient for a multimodular composite. These indicators are necessary to design structural elements made from composites. The object of the study is a unidirectional fibrous composite consisting of the isotropic elastic matrix and fibers. To determine the effective elastic constants, an approach has been suggested, which is based on the use of conditions for the alignment of the displacements of points in the homogenized composite, matrix, and fiber.

First, the displacements and stresses are determined for the matrix and fiber points at their joint axisymmetric stretching. An equation from the multimodular theory of elasticity was preliminarily obtained for solving this problem. Similar components of the stressed-strained state are determined at the same deformation of the cylindrical cell made from a homogeneous transversally-isotropic composite. The conditions for the displacement alignment, derived in solving the specified problems, are the equality of axial displacements in an arbitrary cross-section of the composite by a plane, parallel to the isotropy plane, and the radial displacements at the surface of the composite cell. The result of applying these conditions is the derived formulae for effective constants – the longitudinal module of elasticity of the first kind and a Poisson coefficient, which express these indicators through the mechanical characteristics of the matrix

and fiber, as well as the proportion of fibers in the composite cell volume. Similar formulae have been obtained for the longitudinal compression.

The derived effective elastic characteristics of a transversally-isotropic composite could be used when calculating the stressed-strained state of the structural elements made from it. In this case, one takes into consideration differences in the values of stresses and deformations under axial stretching and compression.

Keywords: homogenization, multimodular transversally-isotropic composite, stresses, displacement, deformation, effective constant.

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DETERMINING THE FEATURES OF LOADING THE IMPROVED BEARING STRUCTURE OF A PLATFORM WAGON FOR THE TRANSPORTATION OF MILITARY EQUIPMENT (p. 20–26)

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The paper reports the improvement in the bearing structure of a platform wagon that transports military equipment and is involved in artillery fire. A special feature of the platform wagon is the presence of rotary sectors made from a composite material with viscous or elastic-viscous links, which makes it possible to absorb the kinetic energy that is transmitted to the frame when firing from the wagon, as well as enables the loading/unloading of military equipment from its side.

We have investigated the dynamic loading of the bearing structure of a platform wagon for military equipment transportation and combat operations. A mathematical model has been constructed, which takes into consideration the movement of the bearing structure of a platform wagon when firing from it. It has been considered that the platform wagon is loaded with two anti-aircraft guns. The mathematical model was solved in the Mathcad programming environment.

The study was conducted in a flat coordinate system. We have determined the accelerations that act on the bearing structure of a platform wagon. The maximum acceleration rate, in this case, is about 3.6 m/s^2 at bouncing oscillations and 4.0 m/s^2 at galloping oscillations. In other words, considering the proposed technical solutions, the dynamic loading of the bearing structure of a platform wagon in the vertical plane at firing decreases by almost 30%. The magnitude of the acceleration is almost independent of the firing angle in this case.

The derived acceleration values have been taken into consideration in determining the strength indicators for the bearing structure of a platform wagon. Calculation was carried out by the method of finite elements in the CosmosWorks programming environment. The maximum equivalent stresses in the bearing structure of a platform wagon amounted to about 285 MPa; they

are concentrated in the region where the bearing structure rests on the trolley. Consequently, the durability of the bearing structure of a platform wagon is ensured.

Modal analysis of the bearing structure of a platform wagon has been conducted. The values of the natural oscillation frequencies are within allowable limits.

Our research would contribute to designing innovative structures for platform wagons.

Keywords: platform wagon, bearing structure, dynamic loading, modal analysis, combined transportation.

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CONSTRUCTION OF AN ALGORITHM FOR THE SELECTION OF RIGID STOPS IN STEEL-CONCRETE BEAMS UNDER THE ACTION OF A DISTRIBUTED LOAD (p. 27–35)

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An algorithm has been developed to select rigid stops in steel-concrete beams under the action of distributed load. Concrete is connected rigidly to a steel sheet in order to perform the joint operation of the concrete and steel sheet. Such a connection in the beam is provided by rigid stops that prevent shifting efforts in the concrete and steel contact area. The efforts are determined through the turning angles between the two adjacent sections of the beam. A graph-analytical method for determining movements is used to determine the turning angles. In determining the deformations of a steel-concrete beam, the calculation is based on the reduced rigidities of cross-sections.

The purpose of this study is to optimize the structure of a steel-concrete beam by selecting the rational number and arrangement of rigid stops. This optimization would allow a more rational utilization of the structure's material – concrete and steel. That would reduce the cost of operations and the quantity of materials required in the production, installation, and operation of the considered structures.

An earlier proposed algorithm for the selection of rigid stops in steel-concrete beams under the action of a concentrated force has been expanded for the case of an evenly distributed load. When selecting the number of rigid stops, it is assumed that the magnitude of the distributed load acting on a beam, the mechanical characteristics of materials (steel and concrete), as well as the span of the beam and the size of its cross-section, are known. In contrast to the beams with a concentrated force in the middle, where the forces abide by a linear law, in the beams with an evenly distributed load the efforts in a steel strip change in line with a square parabola. Therefore, while the same step has been obtained for stops, it is not possible to achieve a situation at which efforts in all stops have the same value.

Keywords: steel-concrete beam, rigid stop, stop step, effort in a stop, reduced rigidity, graphic-analytical method.

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EXPERIMENTAL AND THEORETICAL BUBBLE GROWTH COMPARISON AT THE INITIAL STAGES OF HORIZONTAL INJECTION (p. 36–44)

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Two-phase liquid-gas injection constitutes an important industrial process that is used in most separators. At the early step of injection, a cylindrical bubble is formed. As time elapses, the bubble shape becomes more complex and very difficult to analyze. In this study, a simple analytical model is developed to explain bubble shape changes. The analytical model was developed based on water flow inertia that continually pushes the bubble while the drag force resists it so that the frontal area of the bubble increases. The bubble size and frontal area were estimated using the assumption of the equilibrium between inertia force and drag force neglecting viscous force. From the estimation, the role of the vortex ring from the difference between theoretical and experimental results can be identified. The analytical model was verified through experimental data collected on the shape deformation induced by bubble motion at the beginning of injection. The experimental data used as verification were measured from the bubble nose image with ten times repetition having the uncertainty of $\pm 6\%$. The experimental method is conducted by injecting a bubble along the horizontal direction into a water pool. The inertial force of the water flow in front of the bubble nose generates the bubble. The bubble suddenly changes its shape, moves in the form of a bubble jet, and undergoes gradual shape changes. The frontal area of the bubble increases and reaches a maximum at the terminal velocity point. The bubble shape deformation is affected by the inertial force of the water flow that pushes the bubble forward. Accordingly, the bubble changes its shape from cylindrical to spherical, and then to an ellipsoidal disk. When the bubble attains terminal velocity, the inertial force becomes equal to the drag force. The edge of the ellipsoidal disk bubble exhibits increased surface tension. The difference between experimental data and the analytical model

is due to the complex fluid and dynamic flow surrounding the bubble. The mathematical framework proposed in this work is envisaged to be an important tool for the prediction of the bubble frontal area.

Keywords: injection, deformation, bubble shape, frontal area.

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A ROBOMECH CLASS PARALLEL MANIPULATOR WITH THREE DEGREES OF FREEDOM (p. 44–56)**Zh. Baigunchekov**Al-Farabi Kazakh National University, Almaty,
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This paper presents the methods of structural-parametric synthesis and kinematic analysis of a parallel manipulator with three degrees of freedom working in a cylindrical coordinate system. This parallel manipulator belongs to a RoboMech class because it works under the set laws of motions of the end-effector and actuators, which simplifies the control system and improves its dynamics. Parallel manipulators of a RoboMech class work with certain structural schemes and geometrical parameters of their links. The considered parallel manipulator is formed by connecting the output point to a base using one passive and two active closing kinematic chains (CKC). Passive CKC have zero degree of freedom and it does not impose a geometrical constraint on the movement of the output point, so the geometrical parameters of the links of the passive CKC are freely varied. Active CKCs have active kinematic pairs and they impose geometrical constraints on the movement of the output point. The geometrical parameters of the links of the active CKCs are determined on the basis of the approximation problems of the Chebyshev and least-square approximations. For this, the equations of geometrical constraints are derived in the forms of functions of weighted differences, which are presented in the forms of generalized (Chebyshev) polynomials. This leads to linear iterative problems.

The direct and inverse problems of the kinematics of the investigated parallel manipulator are solved. In the direct kinematics problem, the coordinates of the output point are determined by the given position of the input links. In the inverse kinematics problem, the positions of the input links are determined by the coordinates of the output point. The direct and inverse problems of the kinematics of the investigated parallel manipulator are reduced to solving problems on the positions of Sylvester dyads. Numerical results of structural-parametric synthesis and kinematic analysis of the considered parallel manipulator are presented. The numerical results of the kinematic analysis show that the maximum deviation of the movement of the output point from the orthogonal trajectories is 1.65 %.

Keywords: parallel manipulator, RoboMech, cylindrical coordinate systems, Chebyshev and least-square approximations.

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DESIGNING THE STRUCTURES OF DISCRETE SOLID-ALLOY ELEMENTS FOR BROACHING THE HOLES OF SIGNIFICANT DIAMETER BASED ON THE ASSESSMENT OF THEIR STRENGTH (p. 57–65)

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This paper addresses the issues related to designing and estimating the strength of solid-alloy elements in the deforming broaches of significant diameter (exceeding 150 mm) for the developed process of discrete broaching. The tool limit condition was assessed based on two strength criteria: the specific potential energy of shape change and the maximum tangent stresses. Numerical modeling using the finite element method has made it possible to derive the distribution of equivalent stresses in the tool elements and the contact stresses at the surface of the contact between a solid-alloy insert and the body, which enabled the analysis of tool strength under loading. The simulation was performed under a single normal load, which ensured the versatility of the calculation for any contact pressure values. We have derived for-

mulae to calculate the acceptable contact pressure depending on unit load. The effect of the insert protrusion height over the body on the strength of tool elements has been established. We have derived engineering dependences that determine the required magnitude of insert protrusion over the body depending on the ultimate load. An example of calculating the strength of a prefabricated deforming element in the machining of a sleeve made from gray modified cast iron of hardness HB230 has been considered. Our calculations have shown that the deforming element designed for the new technological process corresponds to the conditions of strength, provided the ratio $h_1/h=0.15$ is maintained (where h_1 is the insert height above the body, h is the insert height). The results obtained could be used in engineering calculations when designing the prefabricated tool for discrete deformation, as well as to assess the strength of prefabricated tools, such as cutters, core drills, reamers, when refining external loads.

Keywords: deforming broaching, stressed state, solid alloy, discrete deforming element, element strength.

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THE REFINED STRENGTH CALCULATION AND OPTIMIZATION OF THE INNER GEOMETRY OF CYLINDRICAL BEARING UNITS (p. 66–78)

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Closed bearing units for railway rolling stock shall operate over 800,000 km or during 8 years of operating life (and, in the near future, 1 million km and 10 years) without any maintenance. In order to achieve such high operational indicators, it is necessary, already at the design stage of closed bearing units, to ensure almost absence of wear during the entire specified operating life.

This paper reports the results of the optimal design of the elements in the internal geometry of closed bearings based on refined mathematical models using an example of the cylindrical axlebox bearing unit “DUPLEX” for 1520 gauge rolling stock. The chosen principal mathematical model was a geometrically nonlinear contact problem from the theory of elasticity, which was solved using a finite element method.

An original non-linear finite-element model of the multi-contact problem has been developed, taking into consideration the contact deformations “rail–wheel”, the deformation of a wheel-set axis, the deformation of the axlebox and bearing rings during contact with all rollers. The model makes it possible to clarify the distribution of loads in the circumferential direction and, accordingly, the maximum load on a roller. The same model could be used, among other things, to analyze the wear of a wheel flange and the effect of gap difference on the bearing wear.

A mathematical model and an objective function have been constructed to optimize the profile of a roller (“crown”, the generatrix of the lateral surface of rotation) considering the ac-

cumulation of damage as a result of the “irregular” loading the roller surface points due to contacts with both the outer and inner rings.

The shapes of the roller’s face and the ring’s operating flange have been optimized that are in contact in the axial direction, which has helped establish that the “anthropologically shaped” convex face of the roller and the concave flange of the ring are optimal. To simplify the structure technologically, a variant with a conical surface of the flange with the optimal “camber” value has been accepted instead of the concave flange of the ring.

Keywords: closed bearing unit, crown, roller, multi-contact problem, finite element method, mathematical model.

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STUDYING THE STEADY-STATE VIBRATIONS OF A TWOMASS VIBRATORY MACHINE EXCITED BY A PASSIVE AUTO-BALANCER (p. 79–86)

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Analytical-numerical methods have been applied to investigate the steady-state vibrations of a two-mass vibratory machine with rectilinear translational motion of platforms and a vibration exciter in the form of a ball, a roller, or a pendulum auto-balancer.

A procedure for studying the modes of load jamming has been devised for the systems similar to the one under consideration. The procedure is based on the idea of parametric solution to the problem of finding the frequencies of load jamming and a bifurcation theory of motion.

It has been established that a two-mass vibratory machine has two resonance frequencies of rotor rotation and two corresponding shapes of platform oscillations. The use of the procedure has shown that for the case of small resistance forces, a vibratory machine:

- has five possible modes of load jamming, with the first shape of resonance vibrations of platforms being excited under modes 1 and 2, the second shape – 3 and 4, and, under the mode 5, the frequency of load jamming is close to the frequency of rotor rotation;

- demonstrates stable jamming modes under the odd (1, 3, 5) load jamming modes;

- shows that the jamming modes 1 and 2 are suitable to excite the resonance oscillations of platforms and for industrial application;

- exhibits that increasing the rotor speed monotonously increases the amplitudes of platform oscillations corresponding to a certain jamming mode;

- proves that the amplitude of resonance platform oscillations can be controlled by changing the rotor rotation velocity.

The viscous resistance forces acting on a first platform reduce (up to the complete elimination) the first range of rotor speeds, at which the first resonance shape of platform oscillations is excited.

The internal forces of viscous resistance, acting between the platforms, reduce (up to the complete elimination) the second range of rotor speeds, at which the second shape of resonance platform oscillations is excited.

The viscous resistance forces acting on the loads at motion relative to an auto-balancer reduce both ranges.

Keywords: inertial vibration exciter, two-frequency vibrations, resonance vibratory machine, auto-balancer, two-mass vibratory machine, Sommerfeld effect.

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SUBSTANTIATION OF THE STABILITY OF HAULAGE DRIFTS WITH PROTECTIVE STRUCTURES OF DIFFERENT RIGIDITY (p. 87–96)

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The purpose of the current research is to substantiate the conditions for the stability of haulage drifts when developing steep coal seams.

The process of modeling the stability of haulage drifts has established that the stressed-strained state of side rocks in a coal-producing array that includes workings depends on the physical and mechanical properties of the roof and soil of the worked coal seam, the rigidity of protective structures, and the length of the roof section supported by a protective structure. Increasing the length of the roof section supported by a protective structure, at the minimal rigidity of pliable supports, increases the zone of smooth bending of the side rocks over a haulage drift and decreases the level of their stressed-strained state.

It has been proven that when maintaining mining workings in deep mines, a reduction in the stressed-strained state of the side rocks, when applying the filling of the worked space, occurs as a result of the sealing of the filling massif on which the roof rocks are based when the values of the compaction factor of the source material accept maximum values equal to $k_{comp}=1.5-1.53$. When using artificial pliable protective structures, erected above a drift, a change in the stressed-strained state occurs as a result of compression of the supports, when the movement of rocks of the roof and soil is limited and the area of contact between side rocks and the erected protective structures increases.

When choosing a protection technique for haulage drifts, it is necessary to take into consideration the parameters of the protective structures, because the impact of the size of the same supports, at the same rigidity, on the distribution of stresses in a coal-rock massif is diverse.

In order to ensure the operational condition of the district preparatory workings in the development of steep coal seams,

it is advisable to use pliable protective structures located above a haulage drift, which limit the movement of side rocks in the worked space.

Keywords: mining pressure, breakage face, side rock collapse, the filling of worked space, pliable supports.

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